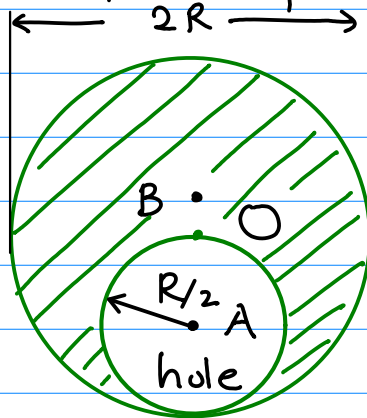


The Cam and shaft problem.

The cross section of the cam is



First let us find its center of mass, located at B. The Mass = M and area of cam

$$A_{\text{cam}} = \pi R^2 - \pi \left(\frac{R}{2}\right)^2 = \frac{3\pi R^2}{4} \quad (1)$$

=> areal density $\rho_A = \frac{4M}{3\pi R^2} \quad (2)$

Think of the cam as composed of a solid disk of density ρ_A , radius R , and mass

$$M_+ = \frac{4}{3} M \quad (3) \quad (4) \quad (5)$$

and another disk of density $-\rho_A$, radius $\frac{R}{2}$ and mass

$$M_- = -\frac{M}{3} \quad (6) \quad (7) \quad (8)$$

located with its center at A

Together, these two disks make up the cam

Choose the origin at O

$$\Rightarrow \boxed{X_{+,CM} = Y_{+,CM} = 0} \quad (9) \quad \begin{array}{l} \text{(CM coordinates)} \\ \text{of } M_+ \end{array}$$

$$\boxed{X_{-,CM} = 0 \quad Y_{-,CM} = -\frac{R}{2}} \quad (10) \quad \begin{array}{l} \text{(CM of)} \\ M_- \end{array}$$

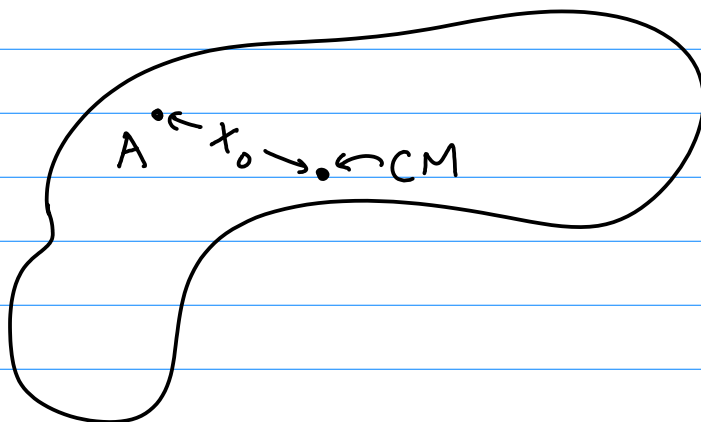
So $X_{CM} = 0$ for the cam

$$\boxed{Y_{CM} = \frac{M_+ Y_{+,CM} + M_- Y_{-,CM}}{M_+ + M_-}} \quad (11)$$
$$= \frac{0 - \left(\frac{M}{3}\right) \left(-\frac{R}{2}\right)}{\frac{4M}{3} - \frac{M}{3}} = \frac{R}{6}$$

So B is $\frac{R}{6}$ above O.

Now find the moment of inertia of the cam about O.

We will need the parallel axis theorem



$$\boxed{I_A = I_{CM} + Mx_0^2} \quad (12)$$

$$\underline{I_0 = I_{+,0} + I_{-,0}} \quad (13)$$

$$\underline{I_{+,0} = \frac{1}{2} M_+ R^2 = \frac{2}{3} MR^2} \quad (14)$$

$$\underline{I_{-,0} = I_{-,cm} + M_- \left(\frac{R}{2}\right)^2} \quad (15) \quad \text{parallel axis thm}$$
$$= \frac{1}{2} \left(-\frac{M}{3}\right) \left(\frac{R}{2}\right)^2 + \left(-\frac{M}{3}\right) \left(\frac{R}{2}\right)^2$$

$$\underline{I_{-,0} = -\frac{M}{3} \frac{R^2}{4} \frac{3}{2} = -\frac{MR^2}{8}} \quad (16)$$

$$\text{So } \underline{I_0 = \frac{2}{3} MR^2 - \frac{MR^2}{8} = \frac{MR^2}{24} (16-3) = \frac{13 MR^2}{24}} \quad (17)$$

Now we need to find the moment of inertia of the cam about its CM, which is B.

Use the parallel axis thm again

$$\underline{I_0 = I_B + M \left(\frac{R}{6}\right)^2} \quad (18)$$

$$\Rightarrow I_B = I_0 - M \left(\frac{R}{6}\right)^2 = MR^2 \left[\frac{13}{24} - \frac{1}{36} \right]$$

$$\underline{I_B = \frac{MR^2}{72} 37} \quad (19)$$

Finally, we need I_A , the moment of inertia of the cam around A. Use the parallel axis thm again

$$\begin{aligned} I_A &= I_B + M \left(\frac{R}{2} + \frac{R}{6} \right)^2 & (20) \\ &= MR^2 \frac{37}{72} + MR^2 \frac{4}{9} = MR^2 \frac{69}{72} = MR^2 \frac{23}{24} \end{aligned}$$

To this one has to add the moment of inertia of the shaft about its axis of symmetry. = $\frac{1}{2} M \left(\frac{R}{2} \right)^2 = \frac{MR^2}{8}$ (21)

$$I_{\text{tot}} = MR^2 \frac{23}{24} + MR^2 \frac{1}{8} = MR^2 \frac{26}{24} = MR^2 \frac{13}{12} \quad (22)$$

So total $KE = \frac{1}{2} I_{\text{tot}} \omega^2 = \frac{13}{24} MR^2 \omega^2$ (23)